

A Stabilized Finite Element Method for the Mixed Wave Equation in an ALE Framework With Application to Diphthong Production

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Summary

Working with the wave equation in mixed rather than irreducible form allows one to directly account for both, the acoustic pressure field and the acoustic particle velocity field. Indeed, this becomes the natural option in many problems, such as those involving waves propagating in moving domains, because the equations can easily be set in an arbitrary Lagrangian-Eulerian (ALE) frame of reference. Yet, when attempting a standard Galerkin finite element solution (FEM) for them, it turns out that an inf-sup compatibility constraint has to be satisfied, which prevents from using equal interpolations for the approximated acoustic pressure and velocity fields. In this work it is proposed to resort to a subgrid scale stabilization strategy to circumvent this condition and thus facilitate code implementation. As a possible application, we address the generation of diphthongs in voice production.

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1. Introduction

The wave equation in mixed form refers, in classical acoustics, to the continuity and momentum equations obtained after linearization of the compressible Navier-Stokes equations for an inviscid homentropic fluid. In contrast to the wave equation in irreducible form, it allows one to describe both, the propagation of acoustic pressure waves and of the acoustic particle velocity as well. This becomes of importance in many different topics which range from intensimetry techniques [1], acoustic wave propagation in ducts [2] or the computation of radiation impedances [3]. Moreover, the wave equation in mixed form also allows one to describe related phenomena in other areas of physics such as elastodynamics or the behavior of waves in shallow waters without convection (see e.g., [4]).

In this work we are interested in numerically solving the problem of wave propagation in moving domains. In particular, our final goal is to be able to produce diphthongs. Though many computational efforts have been placed to solve the fluid-structure problem of human phonation (see e.g., [5, 6, 7, 8, 9, 10, 11] among many others), to the best of our knowledge little has been done with regard to vocal tract acoustics in moving domains, as it is the case for diphthong or syllable generation. Generating diphthongs requires solving the wave equation in a human vocal tract

that evolves from the shape, say for instance of vowel /a/, to the shape of vowel /i/, thus producing the sound /ai/. It is to be noted that for the numerical generation of vowels and diphthongs the acoustic pressure is the variable of interest, so most works to date concerning the generation of vowels directly deal with the FEM solution of the irreducible wave equation [12, 13, 14, 15, 16], or with its time Fourier counterpart, the Helmholtz equation [17, 18, 19, 20]. Occasionally, the wave equation in mixed form has also been solved for vowel generation [21]. In fact, it will be shown that the latter becomes the most natural option when dealing with acoustic wave propagation in moving domains. In such cases the flow continuity and momentum equations have to be expressed in an Arbitrary Lagrangian Eulerian (ALE) [22, 23] frame of reference, rather than in a pure Eulerian one, and this is not straightforward for the irreducible wave equation.

From a numerical and analytical point of view, there are some important differences between the mixed wave equation and its ALE counterpart. Their FEM solutions rely on discretizing the corresponding weak forms. In the case of the variational mixed wave equation, a compatibility inf-sup condition has to be satisfied for the problem to be well-posed. This condition is not directly inherited by the standard Galerkin FEM approach and it becomes necessary to make use of different interpolation spaces for the acoustic pressure and particle velocity to attain a solution free of spurious oscillations [24, 25, 26, 27]. Alternatively, one can resort to stabilized FEM methods, such as the residual based multiscale methods [28, 29, 30], which allow one

to use the same interpolation for the acoustic pressure and particle velocity approximations [4]. A detailed numerical analysis reveals that in this case the solution becomes bounded by the problem data in a new energy norm that contains all designed stabilization terms [31].

The situation becomes more intricate for the ALE wave equation in mixed form. Except for a sign in the mesh velocity term, the mixed ALE wave equation is analogous to the mixed wave equation for waves propagating in a non uniform mean flow. The latter arises for instance, when neglecting the reaction terms in the acoustic perturbation equations (APE) [32], or in some simplified models of the linearized Euler equations (LEE) [33]. In [34], the mixed wave equation with constant convection was addressed as a particular case of the APE. A stabilizing flux term (as in the discontinuous Galerkin method) and a penalization term were incorporated in the formulation to avoid spurious numerical oscillations. The discontinuous Galerkin FEM method was also used in [33] to deal with the simplified LEE. It is also worth noting that the ALE wave equation has also the same mathematical structure as the linearized modified Boussinesq equation for shallow waters with convection. Numerical experiments show that the Galerkin FEM solution for the latter exhibits, once more, strong high-frequency oscillations if equal interpolations are used for the gravity waves depth and velocity [35, 36]. Again these oscillations can be overcome if one draws on stabilization strategies [37, 38]. In this work we will present a subgrid scale stabilization FEM approach tailored to the ALE wave equation in mixed form, following the main lines in [38]. It will be shown that the proposed strategy will allow us to use equal interpolations for the acoustic pressure and velocity, and prevent the FEM solution to blow up. It is to be noted however, that contrary to the variational mixed wave equation, little is known with regard to the well-posedness of the variational continuous and discrete ALE mixed wave equation. Actually, and as far as the authors know, to date only very partial results have been proved showing that the continuous linearized two-dimensional shallow water equations (and by extension the ALE mixed wave equation) are well posed, but only for some specific boundary conditions in a very simple rectangular geometry [39].

This paper is organized as follows. In Section 2 the wave equation in mixed form is set in an ALE framework and its variational formulation is derived. In Section 3, the proposed stabilized FEM approach to solve the latter is presented. Special emphasis is placed on the design of the so called matrix of stabilization parameters. Once presented the fully discrete problem in time and space, some numerical examples are provided in Section 4. First, a test is made to check the performance of the stabilized formulation. It consists of plane waves propagating in a straight duct with time varying length due to the periodic movement of the exit boundary. Second, an example with known analytical solution is presented to check the convergence rate of the method. Finally, a more applied example consisting in the production of diphthong /ai/ is presented. Not only waves

can be visualized propagating inside the vocal tract and exhibiting the right spectrogram, but also an audio file corresponding to /ai/ can be generated by collecting the pressure time evolution at a target node.

2. Problem statement

2.1. The mixed wave equation in an ALE framework

The linearized continuity and momentum equations that describe sound propagation in a stationary inviscid fluid read, in a spatial (Eulerian) frame of reference,

$$\frac{1}{\rho_0 c_0^2} \partial_t p + \nabla \cdot \mathbf{u} = Q, \quad (1a)$$

$$\rho_0 \partial_t \mathbf{u} + \nabla p = \mathbf{f}, \quad (1b)$$

where $p(\mathbf{x}, t)$ stands for the acoustic pressure and $\mathbf{u}(\mathbf{x}, t)$ for the acoustic particle velocity. $Q(\mathbf{x}, t)$ represents a volume source distribution and $\mathbf{f}(\mathbf{x}, t)$ an external body force per unit volume. The parameters c_0 and ρ_0 respectively denote the speed of sound and the mean air density, and ∂_t is used to denote the first order time derivative. Equation (1) is sometimes referred to as the wave equation in mixed form. Its standard irreducible counterpart for the acoustic pressure can be derived subtracting the divergence of (1b) from ρ_0 times the time derivative of (1a). This results in

$$\frac{1}{c_0^2} \partial_{tt}^2 p - \nabla^2 p = \rho_0 \partial_t Q - \nabla \cdot \mathbf{f}. \quad (2)$$

In this work we are interested in solving the mixed wave equation (1) in a domain with moving boundaries. This poses some problems to the Eulerian description of the equations given that one has to take into account the relative movement between the fluid particles and the moving domain. Moreover, dealing with the dynamic boundaries is not straightforward in the Eulerian framework and requires some special approaches such as space-time finite element methods. Yet, setting the mixed wave equation in a pure Lagrangian frame of reference is not exempt of difficulties because computationally expensive mesh rezoning becomes often necessary. An approach that provides a good balance overcoming some of the complications of both descriptions is the so called quasi-Eulerian ALE formulation (see e.g., [40] and references therein). In this case, the time derivative of any fluid property, say g , is expressed in a referential frame moving with the domain, while the spatial derivatives of g are kept in an Eulerian frame [22, 23]. To obtain the ALE expressions for the continuity and momentum equations in (1), we proceed by replacing

$$\partial_t g \leftarrow \partial_t g - \mathbf{u}_d \cdot \nabla g \quad (3)$$

in them, with $\mathbf{u}_d(\mathbf{x}, t)$ denoting the domain velocity. This provides the ALE wave equation in mixed form

$$\frac{1}{\rho_0 c_0^2} \partial_t p - \frac{1}{\rho_0 c_0^2} \mathbf{u}_d \cdot \nabla p + \nabla \cdot \mathbf{u} = Q, \quad (4a)$$

$$\rho_0 \partial_t \mathbf{u} - \rho_0 \mathbf{u}_d \cdot \nabla \mathbf{u} + \nabla p = \mathbf{f}. \quad (4b)$$

In practice, when the equations get discretized $\mathbf{u}_d(\mathbf{x}, t)$ corresponds to the interpolation from the node mesh velocities, so $\mathbf{u}_d(\mathbf{x}, t)$ will be hereinafter referred to as the mesh velocity.

It should be noted that if one attempts at combining (4a) with (4b) to get an ALE irreducible equation for the acoustic pressure, following the procedure employed to obtain (2) but using the relation (3) instead, it is not possible to get rid of the acoustic particle velocity as one obtains

$$\begin{aligned} & \frac{1}{c_0^2} \partial_{tt}^2 p - \nabla^2 p - \frac{1}{c_0^2} \partial_t \mathbf{u}_d \cdot \nabla p - \frac{2}{c_0^2} \mathbf{u}_d \cdot \partial_t \nabla p \\ & + \frac{1}{c_0^2} \mathbf{u}_d \cdot \nabla (\mathbf{u}_d \cdot \nabla p) + \rho_0 (\nabla \mathbf{u}_d)^T : \nabla \mathbf{u} \\ & = \rho_0 \partial_t Q - \rho_0 \mathbf{u}_d \cdot \nabla Q - \nabla \cdot \mathbf{f}. \end{aligned} \quad (5)$$

Therefore, the most natural option to deal with wave propagation in moving domains seems to be that of directly dealing with the wave equation in mixed form. Though it is possible to factorize the wave equation (2) as

$$\left(-\frac{1}{c_0^2} \partial_{tt}^2 p + \nabla^2 p \right) = \left(\frac{1}{c_0} \partial_t + \nabla \right) \left(-\frac{1}{c_0} \partial_t + \nabla \right) p,$$

one should be prevented from substituting (3) in it. As explained at the beginning of this section, equation (2) is obtained from the mass and momentum conservation equations (1a) and (1b) expressed in an Eulerian frame of reference, not an ALE one. It would then be erroneous to directly use (3) in it.

2.2. Variational formulation

Let us next suppose that (4) is to be solved in a computational domain Ω with boundary $\partial\Omega$. $\partial\Omega$ is made of the union of two disjoint sets Γ_p and Γ_u , the pressure being prescribed on the former and the velocity on the latter. For the ease of exposition we will consider homogeneous Dirichlet conditions on them, i.e., $p = 0$ on Γ_p , $t > 0$ and $\mathbf{u} \cdot \mathbf{n} = 0$ on Γ_u , $t > 0$. We also consider initial conditions $p(\mathbf{x}, 0) = p^0(\mathbf{x})$ and $\mathbf{u}(\mathbf{x}, 0) = \mathbf{u}^0(\mathbf{x})$ in Ω .

The weak formulation of (4) is found as usual by multiplying (4a) with a scalar test function q , (4b) with a vector test function \mathbf{v} , and integrating over the computational domain Ω . If we denote the integral of the product between two arbitrary functions f and g as $(f, g) := \int_{\Omega} f g \, d\Omega$, assuming \mathbf{u}_d to be smooth enough, the variational problem can be formulated as that of finding $p \in C^1([0, T], L^2(\Omega)) \cap C^0([0, T], V_p(\Omega))$ and $\mathbf{u} \in C^1([0, T], L^2(\Omega)) \cap C^0([0, T], \mathbf{V}_u(\Omega))$ such that

$$\begin{aligned} & \frac{1}{\rho_0 c_0^2} (\partial_t p, q) - \frac{1}{\rho_0 c_0^2} (\mathbf{u}_d \cdot \nabla p, q) + (\nabla \cdot \mathbf{u}, q) \\ & = (Q, q) \quad \forall q \in V_p(\Omega), \end{aligned} \quad (6a)$$

$$\begin{aligned} & \rho_0 (\partial_t \mathbf{u}, \mathbf{v}) - \rho_0 (\mathbf{u}_d \cdot \nabla \mathbf{u}, \mathbf{v}) + (\nabla p, \mathbf{v}) \\ & = (\mathbf{f}, \mathbf{v}) \quad \forall \mathbf{v} \in \mathbf{V}_u(\Omega), \end{aligned} \quad (6b)$$

with $V_p(\Omega) = \{p \in H^1(\Omega) | p = 0 \text{ on } \Gamma_p\}$ and $\mathbf{V}_u(\Omega) = \{\mathbf{v} \in \mathbf{L}^2(\Omega) | \nabla \cdot \mathbf{v} \in L^2(\Omega), \mathbf{u}_d \cdot \nabla \mathbf{v} \in L^2(\Omega), \mathbf{v} \cdot \mathbf{n} = 0 \text{ on } \Gamma_u\}$. Note that (\cdot, \cdot) may either represent a scalar product in $L^2(\Omega)$ (also in $\mathbf{L}^2(\Omega)$), or a duality pairing depending e.g., on the functional setting of the forcing terms.

With regard to the variational formulation in (6), it should be noted that:

- The Dirichlet boundary conditions (homogeneous in this case) for the acoustic pressure and velocity in (6) are to be strongly imposed on $\partial\Omega$. One could alternatively weakly impose the pressure and strongly impose the velocity integrating by parts the term $(\nabla p, \mathbf{v})$, or viceversa, integrating the term $(\nabla \cdot \mathbf{u}, q)$. If the ALE terms are missing ($\mathbf{u}_d = \mathbf{0}$), the standard wave equation in mixed form is recovered. Integration of the terms $(\nabla p, \mathbf{v})$ or $(\nabla \cdot \mathbf{u}, q)$ noticeably changes in that case the functional framework where the solution for the acoustic pressure and particle velocity are to be found. When attempting a numerical approximation to the corresponding variational forms by means of FEM, this allows one either to obtain the same degree of accuracy for the pressure and the velocity, or to favor one of them at the expense of the other [31].
- As quoted in the Introduction, the ALE wave equation in mixed form, (6), closely resembles the linearized modified Boussinesq equation for shallow waters. The Galerkin FEM solution for the latter is known to suffer from strong high-frequency oscillations [35, 36] when using equal interpolations for the gravity waves depth and velocity, but these can be overcome if one relies on stabilization strategies [37, 38]. These stabilization methods also serve to deal with the convective nature of the terms arising from the ALE formulation. Our next purpose will thus consist in designing an equivalent stabilization for the ALE mixed wave equation.

3. Subgrid scale stabilized finite element method

3.1. Spatial discrete variational form

In this section we will present a subgrid scale stabilization FEM approach for the ALE wave equation in mixed form, based on the variational multiscale method [28, 29]. This will avoid locking effects when using the same interpolations for the acoustic pressure and velocity, as well as accounting for the treatment of the convective terms arising from the ALE formulation. To do so we will closely follow the main lines in previous works on stabilization of the wave equation in mixed form [4] and on the modified Boussinesq equations [38].

First of all, let us rewrite problem (4) in compact matrix form as

$$\boldsymbol{\mu} \partial_t \mathbf{U} + \mathbf{A}_i \partial_i \mathbf{U} = \mathbf{F}, \quad (7)$$

where the summation convention over repeated indexes is assumed when they run from 1 to the number of spatial

dimensions, which we assume to be 3 in what follows. The following identifications have been made,

$$\begin{aligned}
 \mathbf{U} &= \begin{pmatrix} p \\ u_1 \\ u_2 \\ u_3 \end{pmatrix}, \quad \mathbf{F} = \begin{pmatrix} Q \\ f_1 \\ f_2 \\ f_3 \end{pmatrix}, \quad \boldsymbol{\mu} = \begin{pmatrix} \mu_p & 0 & 0 & 0 \\ 0 & \mu_u & 0 & 0 \\ 0 & 0 & \mu_u & 0 \\ 0 & 0 & 0 & \mu_u \end{pmatrix}, \\
 \mathbf{A}_1 &= \begin{pmatrix} -\mu_p u_{d1} & 1 & 0 & 0 \\ 1 & -\mu_u u_{d1} & 0 & 0 \\ 0 & 0 & -\mu_u u_{d1} & 0 \\ 0 & 0 & 0 & -\mu_u u_{d1} \end{pmatrix}, \\
 \mathbf{A}_2 &= \begin{pmatrix} -\mu_p u_{d2} & 0 & 1 & 0 \\ 0 & -\mu_u u_{d2} & 0 & 0 \\ 1 & 0 & -\mu_u u_{d2} & 0 \\ 0 & 0 & 0 & -\mu_u u_{d2} \end{pmatrix}, \\
 \mathbf{A}_3 &= \begin{pmatrix} -\mu_p u_{d3} & 0 & 0 & 1 \\ 0 & -\mu_u u_{d3} & 0 & 0 \\ 0 & 0 & -\mu_u u_{d3} & 0 \\ 1 & 0 & 0 & -\mu_u u_{d3} \end{pmatrix}. \quad (8)
 \end{aligned}$$

To ease the notation and to facilitate comparisons with the preceding work in [4] as well, we have introduced the parameters $\mu_p \equiv (\rho_0 c_0^2)^{-1}$ and $\mu_u \equiv \rho_0$ which fulfill $c_0 = (\mu_p \mu_u)^{-1/2}$. u_{di} , $i = 1, 2, 3$, denote the components of the mesh velocity vector at each point, $\mathbf{u}_d = (u_{d1}, u_{d2}, u_{d3})^\top$.

The variational formulation of (7) is found as usual, multiplying by a test function \mathbf{V} and integrating over the computational domain. If we define the spaces $\mathcal{L} \equiv L^2(\Omega) \times L^2(\Omega)$ and $\mathcal{V} \equiv V_p(\Omega) \times \mathbf{V}_u(\Omega)$, the problem becomes that of finding $\mathbf{U} \in C^1([0, T], \mathcal{L}) \cap C^0([0, T], \mathcal{V})$ such that

$$(\boldsymbol{\mu} \partial_t \mathbf{U}, \mathbf{V}) + (\mathbf{A}_i \partial_i \mathbf{U}, \mathbf{V}) = (\mathbf{F}, \mathbf{V}) \quad \forall \mathbf{V} \in \mathcal{V}. \quad (9)$$

The discretized conforming Galerkin FEM approach to the problem then consists in finding a finite element solution $\mathbf{U}_h \in C^1([0, T], \mathcal{L}_h \subset \mathcal{L}) \cap C^0([0, T], \mathcal{V}_h \subset \mathcal{V})$ that satisfies

$$(\boldsymbol{\mu} \partial_t \mathbf{U}_h, \mathbf{V}_h) + (\mathbf{A}_i \partial_i \mathbf{U}_h, \mathbf{V}_h) = (\mathbf{F}, \mathbf{V}_h) \quad \forall \mathbf{V}_h \in \mathcal{V}_h. \quad (10)$$

Here, \mathcal{L}_h and \mathcal{V}_h are finite spaces constructed from a finite element partition $\{\Omega^e\}$ of Ω , with index e ranging from 1 to the number of elements n_{el} . Given the aforementioned problems with the Galerkin FEM formulation (10), a variational multiscale stabilization approach is next introduced. This consists in splitting the exact solution \mathbf{U} into a finite element component \mathbf{U}_h that can be resolved by the computational mesh, plus a subscale \mathbf{U}' which cannot be computed, and has to be somehow modeled. Substitution of $\mathbf{U} = \mathbf{U}_h + \mathbf{U}'$ into (10) yields two equations; the first one accounting for the large solvable scales is given by

$$\begin{aligned}
 &(\boldsymbol{\mu} \partial_t \mathbf{U}_h, \mathbf{V}_h) + (\boldsymbol{\mu} \partial_t \mathbf{U}', \mathbf{V}_h) \\
 &+ (\mathbf{A}_i \partial_i \mathbf{U}_h, \mathbf{V}_h) + (\mathbf{A}_i \partial_i \mathbf{U}', \mathbf{V}_h) = (\mathbf{F}, \mathbf{V}_h). \quad (11)
 \end{aligned}$$

Assuming quasi-static subscales (i.e., $\partial_t \mathbf{U}' \approx \mathbf{0}$) and integrating by parts the last term in the l.h.s of (12) we get

$$\begin{aligned}
 &(\boldsymbol{\mu} \partial_t \mathbf{U}_h, \mathbf{V}_h) + (\mathbf{A}_i \partial_i \mathbf{U}_h, \mathbf{V}_h) \\
 &+ (\mathbf{U}', \partial_i \mathbf{A}_i^\top \mathbf{V}_h) - (\mathbf{U}', \mathbf{A}_i^\top \partial_i \mathbf{V}_h) = (\mathbf{F}, \mathbf{V}_h). \quad (12)
 \end{aligned}$$

The term $(\mathbf{U}', \partial_i \mathbf{A}_i^\top \mathbf{V}_h)$ plays the role of a reaction term and has been deemed negligible in the numerical experiments of Section 4. Given that it can be discarded without affecting the consistency of the method, it has not been considered hereinafter.

The second equation, governing the behavior of the static subscales, is given by

$$\begin{aligned}
 &(\boldsymbol{\mu} \partial_t \mathbf{U}_h, \mathbf{V}') + (\mathbf{A}_i \partial_i \mathbf{U}_h, \mathbf{V}') \\
 &+ (\mathbf{A}_i \partial_i \mathbf{U}', \mathbf{V}') = (\mathbf{F}, \mathbf{V}'), \quad (13)
 \end{aligned}$$

which is nothing but the L^2 -projection of $\boldsymbol{\mu} \partial_t \mathbf{U}_h + \mathbf{A}_i \partial_i \mathbf{U}_h + \mathbf{A}_i \partial_i \mathbf{U}' = \mathbf{F}$ onto the space of subscales. Using \mathcal{P} to denote this projection, (13) can be rewritten as

$$\mathcal{P}(\mathbf{A}_i \partial_i \mathbf{U}') = \mathcal{P}[\mathbf{F} - (\partial_t \mathbf{U}_h + \mathbf{A}_i \partial_i \mathbf{U}_h)] =: \mathbf{R}_h, \quad (14)$$

where we have defined the residual \mathbf{R}_h of the finite element approximation onto the subscale space in the last equality. As said, (14) is an equation for the subscales \mathbf{U}' whose solution is unknown and has to be modeled. A reasonable approximation is that of taking [4, 38] $\mathcal{P}(\mathbf{A}_i \partial_i \mathbf{U}') \approx \boldsymbol{\tau}^{-1} \mathbf{U}'$, so that from (14) $\mathbf{U}' = \boldsymbol{\tau} \mathbf{R}_h$, with $\boldsymbol{\tau}$ standing for a symmetric, positive-definite matrix of stabilization parameters to be determined. Substituting $\mathbf{U}' = \boldsymbol{\tau} \mathbf{R}_h$ into the large scale equation (12) yields

$$\begin{aligned}
 &(\boldsymbol{\mu} \partial_t \mathbf{U}_h, \mathbf{V}_h) + (\mathbf{A}_i \partial_i \mathbf{U}_h, \mathbf{V}_h) \\
 &+ \sum_{e=1}^{n_{el}} (\mathbf{A}_i^\top \partial_i \mathbf{V}_h, \boldsymbol{\tau} \mathcal{P}[(\partial_t \mathbf{U}_h + \mathbf{A}_i \partial_i \mathbf{U}_h) - \mathbf{F}])_{\Omega_e} = (\mathbf{F}, \mathbf{V}_h). \quad (15)
 \end{aligned}$$

The first two terms in (16) correspond to the standard Galerkin FEM approach in (10), whereas the third one is the additional stabilization term that facilitates using equal interpolations for the acoustic pressure and acoustic particle velocity. Concerning the stabilized variational formulation we note that

- There exist different options for the projection operator \mathcal{P} in the above expression. In the so called OSS (Orthogonal Subgrid Scale) method [30, 41], the subscales are assumed to lay in a space orthogonal to the finite element space so that \mathcal{P} can be computed as $\mathcal{P} = \mathbf{I} - \Pi_h$, Π_h being the L^2 -projection onto the finite element space, and \mathbf{I} the identity matrix. Alternatively, in the more classical ASGS (Algebraic Subgrid Scale) approach we simply take $\mathcal{P} = \mathbf{I}$ over the space of finite element residuals.
- The approximation for the subscales is local and these are assumed to vanish at the interelemental boundaries. The integrals over the computational domain Ω that contain subscales are evaluated as a summation of integrals over finite element domains Ω_e .
- The integrals in the above variational formulations contain products of vector functions having the dimensions $[\mathbf{U}^\top \mathbf{F}]$ ($[\cdot]$ is used to denote a dimensional group). Given that $\mathbf{U}^\top \mathbf{F} = pQ + u_1 f_1 + u_2 f_2 + u_3 f_3$ we require that $[pQ] = [u_i f_i]$. This is the case for the ALE mixed wave equation as it can be readily be checked

that $[pQ] = [u_i f_i] = MLT^{-3}$, M , L and T respectively standing for dimensions of mass, length and time. However, in most equations vector products of the type $U^T F$ are not well defined and a rescaling of the differential equation becomes necessary for the variational form in the compact matrix form to make sense [4, 38].

- To complete the weak formulation (16) it only remains finding an expression for the stabilization matrix τ . This will be the subject of the next subsection.

3.2. The matrix τ of stabilization parameters

The procedure to find the stabilization matrix τ , to be described below, involves dealing with the norms of the residual vector, which has dimensions of force, and of the subscale velocity vector. However, we note that contrary to $U^T F$, products like $F^T F$ or $U^T U$ are not dimensionally well defined. For instance, $F^T F = Q^2 + f_1^2 + f_2^2 + f_3^2$ but $[Q^2] \neq [f_i^2]$. Analogously $U^T U = p^2 + u_1^2 + u_2^2 + u_3^2$ and again, obviously, $[p^2] \neq [u_i^2]$. To correctly define these products we can introduce the weighting matrix [4] M ,

$$\mathbf{M} = \begin{pmatrix} m_p & 0 & 0 & 0 \\ 0 & m_u & 0 & 0 \\ 0 & 0 & m_u & 0 \\ 0 & 0 & 0 & m_u \end{pmatrix},$$

$$m_p := \sqrt{\mu_u/\mu_p} = \rho_0 c_0, \quad (16)$$

$$m_u := \sqrt{\mu_p/\mu_u} = \frac{1}{\rho_0 c_0},$$

and readily check that $F^T M F \equiv |F|_M^2$ is now dimensionally correct because $[m_p Q^2] = [m_u f_i^2] = ML^{-2}T^{-3}$. With regard to unknowns U , we have to use the inverse of the M matrix to get the right dimensions. Indeed, it follows that $U^T M^{-1} U \equiv |U|_{M^{-1}}^2$ is well defined because $[m_p^{-1} p^2] = [m_u^{-1} u_i^2] = MT^{-3}$. Thus we can use M and M^{-1} to respectively define products between any two force vectors $F_1^T M F_2$ and unknowns $U_1^T M^{-1} U_2$. We can proceed similarly and define a M -weighted product between two arbitrary matrices B_1 and B_2 as $B_1^T M B_2$. The squared M -pointwise norm of a matrix B will be given by $|B|_M^2 = \sup\{X^T B^T M B X\}$, $\forall X; |X|_{M^{-1}} = 1$. Equipped with the weighting matrix M and for regular enough force vector functions, we can also define the following vector function scalar products,

$$(U, F) = \int_{\Omega} U^T F \, d\Omega,$$

$$(F_1, F_2)_M := \int_{\Omega} F_1^T M F_2 \, d\Omega, \quad (17)$$

$$(U_1, U_2)_{M^{-1}} := \int_{\Omega} U_1^T M^{-1} U_2 \, d\Omega,$$

whose induced norms will be denoted by $\|\cdot\|$, $\|\cdot\|_M$ and $\|\cdot\|_{M^{-1}}$.

A suitable option to find the matrix of stabilization parameters τ is based on a Fourier analysis of the equation for the subscales (14). Let us use a hat symbol to denote

Fourier transformed functions. Transforming (14) to the wave number domain allows us to bound the norm of the Fourier transformed residual \hat{R}_h as [4, 38]

$$\|\hat{R}_h\|_M^2 \leq \left| -i \frac{1}{h} k_j^0 A_j \right|_M^2 \|\hat{U}'\|_{M^{-1}}^2, \quad (18)$$

with k_j^0 standing for particular dimensionless wave number components that arise from the application of the mean value theorem and that, in practice, will be treated as algorithmic constants (see below). On the other hand, from the Fourier transform of the approximation for the subscales $U' = \tau R_h$, we can get the bound

$$\|\hat{R}_h\|_M^2 \leq |\tau^{-1}|_M^2 \|\hat{U}'\|_{M^{-1}}^2. \quad (19)$$

Therefore, (18) and (19) provide a means to get the stabilization matrix τ by equating $|\tau^{-1}|_M^2$ with $\left| -i \frac{1}{h} k_j^0 A_j \right|_M^2$, i.e.,

$$\sup_{|X|_{M^{-1}}=1} X^T (\tau^{-1} M \tau^{-1}) X$$

$$= \sup_{|X|_{M^{-1}}=1} X^T \left(\frac{1}{h^2} k_i^0 k_j^0 A_i^T M A_j \right) X. \quad (20)$$

In practice we can implement the above condition by computing the spectra of matrices $\tau^{-1} M \tau^{-1}$ and $\frac{1}{h^2} k_i^0 k_j^0 A_i^T M A_j$ with respect to matrix M^{-1} , and then imposing their spectral radii to be the same.

Taking into account the definitions for A_i in (8) and for the scaling matrix M in (17), we get for the problem at hand,

$$\frac{1}{h^2} k_i^0 k_j^0 A_i^T M A_j = \frac{1}{h^2} \cdot \quad (21)$$

$$\begin{pmatrix} m_u |k^0|^2 + \beta_p |k^0 \cdot u_d|^2 & -\alpha k_1^0 (k^0 \cdot u_d) & & & \\ -\alpha k_1^0 (k^0 \cdot u_d) & m_p (k_1^0)^2 + \beta_u |k^0 \cdot u_d|^2 & & & \\ -\alpha k_2^0 (k^0 \cdot u_d) & & k_1^0 k_2^0 m_p & & \dots \\ -\alpha k_3^0 (k^0 \cdot u_d) & & k_1^0 k_3^0 m_p & & \\ & -\alpha k_2^0 (k^0 \cdot u_d) & & -\alpha k_3^0 (k^0 \cdot u_d) & \\ & k_1^0 k_2^0 m_p & & k_1^0 k_3^0 m_p & \\ \dots & m_p (k_2^0)^2 + \beta_u |k^0 \cdot u_d|^2 & & k_2^0 k_3^0 m_p & \\ & k_2^0 k_3^0 m_p & & m_p (k_3^0)^2 + \beta_u |k^0 \cdot u_d|^2 & \end{pmatrix}$$

with

$$\alpha \equiv (m_p \mu_p + m_u \mu_u), \quad \beta_p \equiv m_p \mu_p^2, \quad \beta_u \equiv m_u \mu_u^2. \quad (22)$$

Let us denote the spectrum of the generalized eigenvalue problem $B U = \lambda M^{-1} U$, λ being an eigenvalue, by $\text{Spec}_{M^{-1}}(B)$, and its spectral radius by $\rho(B)$. The spectrum of $k_i^0 k_j^0 A_i^T M A_j$ can be analytically worked out and is given by

$$\text{Spec}_{M^{-1}}(k_i^0 k_j^0 A_i^T M A_j) =$$

$$\left\{ \left(\frac{k^0 \cdot u_d}{c_0} + |k^0| \right)^2, \left(\frac{k^0 \cdot u_d}{c_0} \right)^2, \right. \quad (23)$$

$$\left. \left(\frac{k^0 \cdot u_d}{c_0} \right)^2, \left(\frac{k^0 \cdot u_d}{c_0} - |k^0| \right)^2 \right\},$$

where, as seen, the eigenvalue $c_0^{-2}(\mathbf{k}^0 \cdot \mathbf{u}_d)^2$ has multiplicity 2. As observed, the spectral radius of $k_i^0 k_j^0 \mathbf{A}_i^\top \mathbf{M} \mathbf{A}_j$ is given by

$$\begin{aligned} \rho(k_i^0 k_j^0 \mathbf{A}_i^\top \mathbf{M} \mathbf{A}_j) &= \left(\frac{\mathbf{k}^0 \cdot \mathbf{u}_d}{c_0} + |\mathbf{k}^0| \right)^2 \\ &\equiv \left(\frac{C_1 |\mathbf{u}_d|}{c_0} + C_2 \right)^2. \end{aligned} \quad (24)$$

Since \mathbf{k}^0 is an unknown wave number, in the r.h.s equivalence we have taken $\mathbf{k}^0 \cdot \mathbf{u}_d = |\mathbf{k}^0| |\mathbf{u}_d| \cos \theta \equiv C_1 |\mathbf{u}_d|$ and $|\mathbf{k}^0| \equiv C_2$, C_1 , C_2 being constants that should be determined from numerical experiments.

In order to get a simple expression for the stabilization matrix we may take it to be diagonal, $\boldsymbol{\tau} = \text{diag}(\tau_p, \tau_u, \tau_u, \tau_u)$. Given that the scaling matrix \mathbf{M} in (17) is also diagonal, it follows that

$$\begin{aligned} \text{Spec}_{\mathbf{M}^{-1}}(\boldsymbol{\tau}^{-1} \mathbf{M} \boldsymbol{\tau}^{-1}) &= \\ \left\{ \left(\frac{m_p}{\tau_p} \right)^2, \left(\frac{m_u}{\tau_u} \right)^2, \left(\frac{m_u}{\tau_u} \right)^2, \left(\frac{m_u}{\tau_u} \right)^2 \right\}. \end{aligned} \quad (25)$$

As we want to fulfill the condition

$$\rho(\boldsymbol{\tau}^{-1} \mathbf{M} \boldsymbol{\tau}^{-1}) = \rho(h^{-2} k_i^0 k_j^0 \mathbf{A}_i^\top \mathbf{M} \mathbf{A}_j), \quad (26)$$

we may achieve that by forcing every eigenvalue of $\boldsymbol{\tau}^{-1} \mathbf{M} \boldsymbol{\tau}^{-1}$ in (26) to be equal to (24). This results in the following matrix of stabilization parameters

$$\boldsymbol{\tau} = \begin{pmatrix} \tau_p & 0 & 0 & 0 \\ 0 & \tau_u & 0 & 0 \\ 0 & 0 & \tau_u & 0 \\ 0 & 0 & 0 & \tau_u \end{pmatrix} \quad (27)$$

with

$$\begin{aligned} \tau_p &= \frac{c_0 h \sqrt{\mu_u / \mu_p}}{C_1 |\mathbf{u}_d| + c_0 C_2} = \frac{\rho_0 c_0^2 h}{C_1 |\mathbf{u}_d| + c_0 C_2}, \\ \tau_u &= \frac{c_0 h \sqrt{\mu_p / \mu_u}}{C_1 |\mathbf{u}_d| + c_0 C_2} = \frac{h}{C_1 \rho_0 |\mathbf{u}_d| + \rho_0 c_0 C_2}, \end{aligned}$$

and where use has been made of the expressions for m_p and m_u in (17). It is observed that:

- The matrix $\boldsymbol{\tau}$ of stabilization parameters (27) is to be inserted into (16) to get the space stabilized variational formulation proposed in this work.
- In the case of a static domain $\mathbf{u}_d = \mathbf{0}$ and we recover from (27) the stabilization parameters for the standard wave equation in mixed form [4],

$$\tau_p = Ch \sqrt{\mu_u / \mu_p}, \quad \tau_u = Ch \sqrt{\mu_p / \mu_u}, \quad (28)$$

with C standing for a constant. Depending on the values of μ_p and μ_u the mixed wave equation can be used to describe phenomena in classical acoustics, elastodynamics or wave propagation in shallow waters [4, 38].

3.3. Fully discrete problem

Up to now we have left the time continuous in the previous developments. As for the time discretization, we equally split the time interval $[0, T]$ into N steps $0 < t^1 < t^2 < \dots < t^n < \dots < t^N \equiv T$ such that $\Delta t := t^{n+1} - t^n$ is the time step size. For any time dependent function $g(t)$, g^n will stand for its evaluation at $t^n = n\Delta t$, or an approximation to it. The time evolution of (16) is then approximated using a second order backward differentiation formula (BDF2). Let us identify $\delta_t g^{n+1} := (1/2\Delta t)(3g^{n+1} - 4g^n + g^{n-1})$ so that the time discrete version of (16) becomes

$$\begin{aligned} &(\boldsymbol{\mu} \delta_t \mathbf{U}_h^{n+1}, \mathbf{V}_h) + (\mathbf{A}_i \partial_i \mathbf{U}_h^{n+1}, \mathbf{V}_h) \\ &+ \sum_{e=1}^{n_{el}} (\mathbf{A}_i^\top \partial_i \mathbf{V}_h, \boldsymbol{\tau} \mathcal{P}[(\delta_t \mathbf{U}_h^{n+1} + \mathbf{A}_i \partial_i \mathbf{U}_h^{n+1}) - \mathbf{F}^{n+1}])_{\Omega_e} \\ &= (\mathbf{F}^{n+1}, \mathbf{V}_h). \end{aligned} \quad (29)$$

If the OSS approach is to be implemented, which corresponds to the choice $\mathcal{P} = \mathbf{I} - \Pi_h$, we get $\mathcal{P}(\delta_t \mathbf{U}_h) = 0$. Then, the expanded fully discrete problem that results from taking into account (8) in (29) and using the BDF2 scheme becomes

$$\begin{aligned} &\frac{1}{\rho_0 c_0^2} (\delta_t p_h^{n+1}, q_h) - \frac{1}{\rho_0 c_0^2} (\mathbf{u}_d \cdot \nabla p_h^{n+1}, q_h) + (\nabla \cdot \mathbf{u}_h^{n+1}, q_h) \\ &+ \sum_{e=1}^{n_{el}} \left(\tau_p \mathcal{P} \left[-\frac{1}{\rho_0 c_0^2} \mathbf{u}_d \cdot \nabla p_h^{n+1} + \nabla \cdot \mathbf{u}_h^{n+1} - \mathcal{Q}^{n+1} \right], \right. \\ &\quad \left. - \frac{1}{\rho_0 c_0^2} \mathbf{u}_d \cdot \nabla q_h + \nabla \cdot \mathbf{v}_h \right)_{\Omega_e} = (\mathcal{Q}^{n+1}, q_h), \end{aligned} \quad (30a)$$

$$\begin{aligned} &\rho_0 (\delta_t \mathbf{u}_h^{n+1}, \mathbf{v}_h) - \rho_0 (\mathbf{u}_d \cdot \nabla \mathbf{u}_h^{n+1}, \mathbf{v}_h) + (\nabla p_h^{n+1}, \mathbf{v}_h) \\ &+ \sum_{e=1}^{n_{el}} \left(\tau_u \mathcal{P} \left[-\rho_0 \mathbf{u}_d \cdot \nabla \mathbf{u}_h^{n+1} + \nabla p_h^{n+1} - \mathbf{f}^{n+1} \right], \right. \\ &\quad \left. - \rho_0 \mathbf{u}_d \cdot \nabla \mathbf{v}_h + \nabla q_h \right)_{\Omega_e} = (\mathbf{f}^{n+1}, \mathbf{v}_h). \end{aligned} \quad (30b)$$

The projection \mathcal{P} is understood to be applied to vectors or scalars, depending on the argument. Instead, if one adopts the ASGS stabilization method $\mathcal{P} = \mathbf{I}$ and the terms $\mathcal{P}(\delta_t p_h)$ and $\mathcal{P}(\delta_t \mathbf{u}_h)$ have to be respectively included in Eqs. (30a) and (30b). Equation (29), with the stabilization parameters in (27), is the final fully discrete scheme in space and time proposed in this work, which has been implemented for the numerical examples to be presented in the next section.

3.4. Motion of the computational mesh

With regard to the motion of the computational mesh, it will be prescribed by the known displacement at the domain boundaries. The motion of the boundary nodes becomes smoothly transmitted to the inner mesh nodes through diffusion, i.e. using the standard strategy of solving the Laplacian equation for the node displacements $\mathbf{w}(\mathbf{x}, t)$. Therefore, at each time step of (29) we will solve the additional equation

$$\nabla^2 \mathbf{w}^{n+1} = 0 \quad \text{in } \Omega, \quad t = t^{n+1}, \quad (31a)$$

with boundary conditions

$$\begin{aligned} \mathbf{w}^{n+1} &= \mathbf{x}_{\text{mw}}^{n+1} - \mathbf{x}_{\text{mw}}^n & \text{on } \Gamma_{\text{MB}}, t = t^{n+1}, \\ \mathbf{w}^{n+1} \cdot \mathbf{n} &= 0 & \text{on } \Gamma_{\text{FB}}, t = t^{n+1}, \end{aligned} \quad (31b)$$

the mesh node positions being updated according to

$$\mathbf{x}^{n+1} = \mathbf{x}^n + \mathbf{w}^{n+1}. \quad (31c)$$

Γ_{MB} in (31b) represents that boundary of Ω evolving with prescribed displacement $\mathbf{x}_{\text{mw}}(t)$, whereas Γ_{FB} in (31b) denotes a fixed boundary of Ω . From the displacements $\mathbf{w}(\mathbf{x}, t)$ one can compute the mesh velocity using an appropriate time integration scheme.

4. Numerical examples

4.1. Two dimensional duct with periodically varying length

In order to check the performance of the proposed stabilization strategy, we have adopted a numerical example consisting of a two dimensional duct of constant cross section but varying length, see Figure 1. A periodic velocity $\mathbf{u}_g(t)$ has been imposed at the duct entrance (Γ_G in the figure) which results in the generation of plane waves propagating to the opposite side of the duct. Yet, a periodic displacement of different frequency has been imposed at the exit boundary Γ_M . This guarantees that the mesh velocity \mathbf{u}_d will be parallel to the acoustic pressure gradient. When \mathbf{u}_d is parallel to ∇p , the importance of the ALE terms in (4) becomes maximum so the test can be seen as demanding for the validation of the stabilized method. Besides, note that all duct dimensions, boundary displacements and involved frequencies (see below) have been chosen to be of the same order of that found in real human vocal tracts, lip protrusion movements and formant frequencies. This will guarantee that the stabilization will perform correctly when addressing the production of diphthongs.

The details of the simulation are as follows. We have split the duct boundary into three non-intersecting regions Γ_G , Γ_W and Γ_M (see Figure 1) and thus supplement the ALE mixed wave equation (4) with boundary conditions

$$\mathbf{u} \cdot \mathbf{n} = \mathbf{u}_g(t) = \sin(2\pi f_3 t) \quad \text{on } \Gamma_G, t > 0, \quad (32a)$$

$$\mathbf{u} \cdot \mathbf{n} = 0 \quad \text{on } \Gamma_W, t > 0, \quad (32b)$$

$$p = 0 \quad \text{on } \Gamma_M, t > 0. \quad (32c)$$

It is observed that we have assumed rigid boundaries and imposed a zero pressure release condition at the duct exit. \mathbf{n} denotes the normal pointing outwards the computational boundary. With regard to the velocity on Γ_G , it has a sinusoidal time dependence whose frequency corresponds to the third resonance mode of the mean length duct ($L = 17$ cm). The n -th resonance frequency of such a duct is given by $f_n = (2n - 1)c_0/(4L)$ so by taking $n = 3$, $c_0 = 350$ m/s and $L = 17$ cm we get $f_3 \simeq 2573$ Hz. A periodic displacement of frequency $f_M = 500$ Hz has been imposed for Γ_M . Finally, zero pressure and zero particle

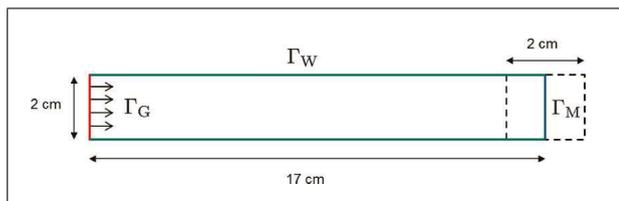


Figure 1. Sketch of the two dimensional duct example used to test the stabilized FEM approach for the ALE mixed wave equation. Γ_G stands for the boundary where the input signal $\mathbf{u}_g(t)$ is introduced, Γ_W the duct walls and Γ_M the duct exit, which moves sinusoidally backward and forward.

velocity have been considered for $t = 0$ inside the duct and a value of $\rho_0 = 1.21 \text{ kg/m}^3$ has been taken for the air density.

The simulations have been carried out using the fully discrete scheme in (29), activating and deactivating the ALE ASGS stabilizations terms. Linear P1/P1 elements have been employed for a mesh with characteristic size $h \sim 0.004$ m. From numerical experiments, the values deemed appropriate for the constants C_1 and C_2 in the expression for the stabilization parameters (27) are $C_1 = C_2 = 0.01$. The stabilization parameters also dictate the numerical critical time step size (see e.g., [41]) which is given in the present work by $\Delta t_c \sim \rho_0 \tau \mathbf{u} = (\rho_0 c_0^2) \tau_p \sim 3.25 \times 10^{-4}$ s. Δt_c involves the mesh size and the physical parameters in the mixed wave equation. The time step size Δt for the implicit BDF2 scheme in subsection 3.3 can be typically relaxed from 5 to 100 times Δt_c , depending on the problem. However, Δt_c is not the only time step to be considered because it does not involve the problem boundary conditions. In the case of a prescribed harmonic motion at the boundary, it is well-known that one should have a resolution of six to ten points per period and wavelength to correctly account for wave propagation. This actually defines another possible time step size, Δt_w , so that the minimum between $s\Delta t_c$ and Δt_w has to be chosen (where s is the factor mentioned earlier, usually between 5 and 100) to run the simulation. For the example at hand $\Delta t_w \sim 4 \times 10^{-5}$ so this is the time step size to be selected. Actually, we have chosen a slightly smaller value of $\Delta t = 1.25 \times 10^{-5}$ for the simulations given that the simple 2D examples in this subsection are not computationally demanding. Besides, we note that the mesh has been updated every Δt following the strategy described in Subsection 3.4.

The obtained results are presented in Figure 2. In the first column we show different snapshots of the acoustic pressure inside the duct when using all stabilization terms. As observed, the inner pressure pattern basically corresponds to the third acoustic mode distribution which does not become much altered by the movement of the exit boundary. The solution is stable and does not change significantly as time evolves. This is not the case if the ALE stabilization terms are removed from the formulation, as can be readily appreciated in the right column of Figure 2. Though the solution matches the correct one for the initial time steps of the simulation, as long as time evolves

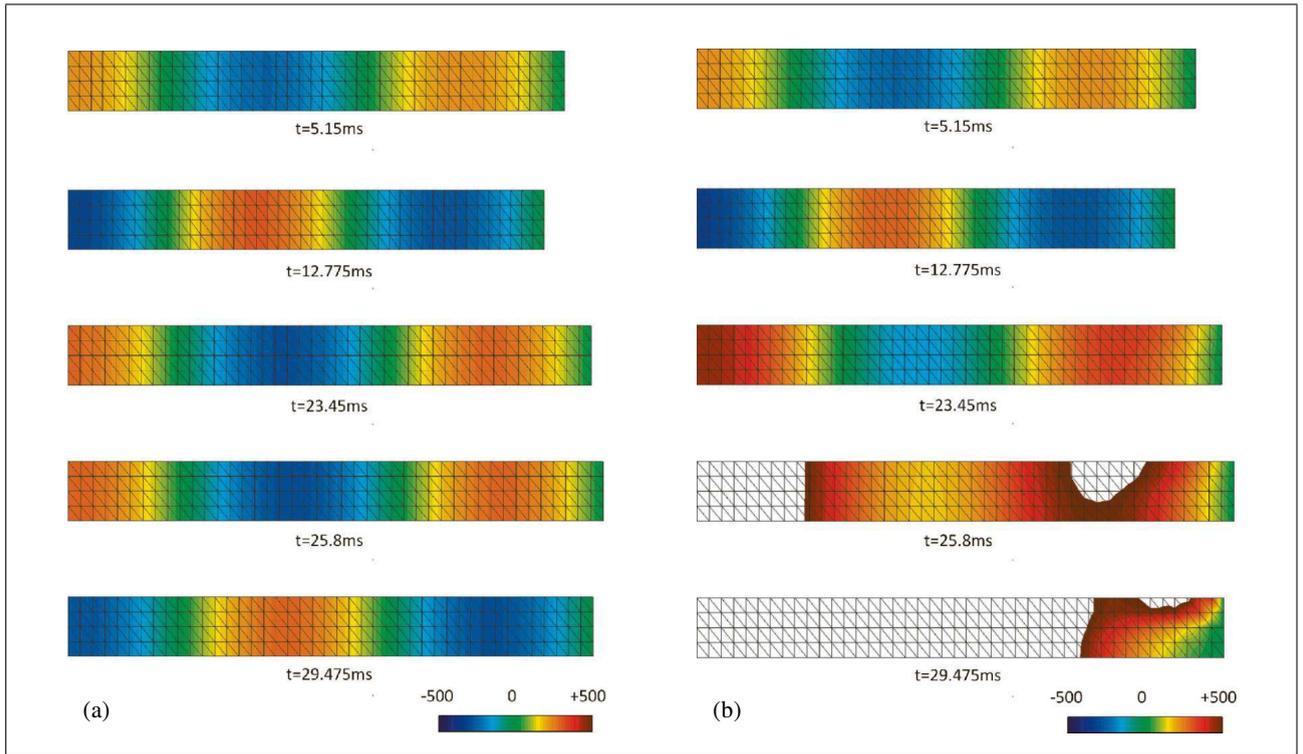


Figure 2. Snapshots of the acoustic pressure in Pa inside the 2D duct at different time steps considering (a) all stabilization terms and (b) removing the ALE stabilization terms.

the solution becomes unstable and finally blows up, totally exceeding the plotted range $[-500, 500]$ Pa.

4.2. Two dimensional case with analytical solution for convergence tests

In the preceding subsection it has been shown how the proposed strategy prevents the instabilities of the standard Galerkin FEM approach. We will next focus on a problem with known analytical solution to check the convergence rate of the FEM error, when refining the computational mesh. The influence of the values of the constants in the stabilization parameters on the convergence curves will be also presented.

To those purposes let us consider the same computational domain as in Figure 1, but now with Γ_M fixed and with homogenous boundary conditions for the pressure on Γ_G and Γ_M , and also $\mathbf{u} \cdot \mathbf{n} = 0$ on Γ_W . The same time step size and P1/P1 elements in the preceding subsection are also used.

We force the following analytical solution that only depends on the x -coordinate and time

$$p_{ex}(\mathbf{x}, t) = \sin(3\pi x/L) \sin(2\pi t), \quad (33a)$$

$$\mathbf{u}_{ex}(\mathbf{x}, t) = [p, 0]^T. \quad (33b)$$

The mesh velocity, which is identified in this example with a flow convection velocity, is taken as $\mathbf{u}_d = [-c_0 x, 0]^T$. We can achieve the exact solution (33) by imposing the subsequent volume source distribution Q and

external force $\mathbf{f} = [f_x, f_y]^T$ in (4),

$$Q(\mathbf{x}, t) = \frac{2\pi}{\rho_0 c_0^2} \sin(3\pi x/L) \cos(2\pi t) + \frac{3\pi}{\rho_0 c_0 L} (x + \rho_0 c_0) \cos(3\pi x/L) \sin(2\pi t), \quad (34a)$$

$$f_x(\mathbf{x}, t) = \rho_0^2 c_0^2 Q + \frac{3\pi}{L} (1 - \rho_0^2 c_0^2) \cos(3\pi x/L) \sin(2\pi t),$$

$$f_y(\mathbf{x}, t) = 0. \quad (34b)$$

In other words, Q and \mathbf{f} in (34) are such that compensate for the convection velocity \mathbf{u}_d so that we can still observe the pressure pattern (33a) inside the duct, even when convection is present. Obviously, this example lacks of physical realism and it is only intended to check the convergence of the method by comparing the exact solution (33), with that of the proposed stabilized FEM approach.

Convergence curves of the time average of the absolute error $[\int_{\Omega} (p_{ex} - p_h)^2 d\Omega]^{1/2}$ and $[\int_{\Omega} (\mathbf{u}_{ex} - \mathbf{u}_h)^2 d\Omega]^{1/2}$ against the mesh size are plotted with a logarithm scale in Figure 3. In Figure 3a we present the results that correspond to the acoustic pressure whereas those for the acoustic particle velocity are shown in Figure 3b. As observed from the figures, changing the values of the constants in the stabilization parameters affects the absolute error but slightly modifies the rate of convergence (see Table I). Note that the experimental rates of convergence are fine. As a point for comparison we could take those for the wave equation in mixed form without ALE terms in [31].

Table I. Experimental convergence rates for the acoustic pressure and velocity depending on the constants of the stabilization parameters.

Stabilization parameters	Pressure rate	Velocity rate
$C_1 = C_2 = 0.005$	2.32	2.09
$C_1 = C_2 = 0.01$	2.57	2.22
$C_1 = C_2 = 0.02$	2.79	2.33

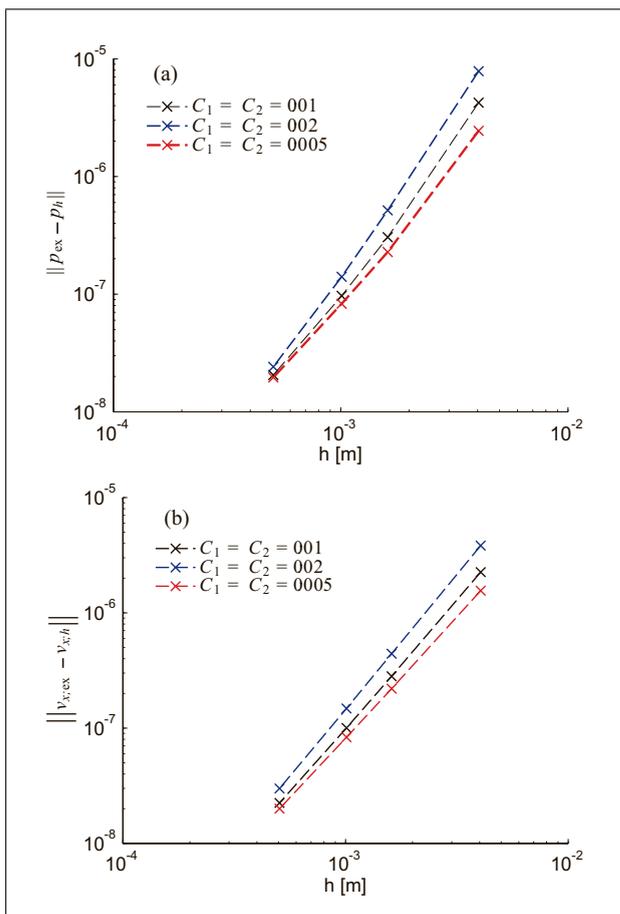


Figure 3. Convergence of the approximation using linear P1/P1 elements (a) pressure field (b) velocity in the x -direction.

The experimental convergence rates for the pressure and velocity in that work were nearly 2, whereas the minimum expected convergence rate predicted theoretically was 1. The herein reported rates are slightly better, partially because the errors have been averaged on time, whereas the maximum norm in time was taken in [31].

4.3. Production of diphthongs in 3D

A more applied example related to voice production will be next addressed. In particular, we will focus on the numerical generation of diphthongs. The diphthong /ai/ will be used for illustrative purposes though the presented procedure has general validity.

Let us start by reviewing how a single vowel could be generated using numerical methods. In a nutshell, to produce say, vowel /a/, we could proceed as follows (see left

column in Figure 4). First a magnetic resonance imaging (MRI) is taken from the vocal tract of a given individual when pronouncing the sound /a/. Though nowadays very detailed 3D geometries can be obtained [42], it is not always necessary to use such intricate shapes for voice generation. Actually we can neglect the influence of some of the vocal tract cavities, such as the valleculae or the piriform fossae, and work with simplified straight vocal tracts with radial symmetry based on real vocal tract cross sections, as those in [43] (see second row in the first column of Figure 4). Despite of its simplicity, fairly good results can still be obtained with these smoothed vocal tract geometries. Since it is not the goal of this work to reproduce diphthongs from a given individual with high fidelity, but rather to propose a general approach for diphthong generation, we will make use of these simplified geometries in the simulations to be presented below. Next, we just need to solve the wave equation for the acoustic pressure inside such vocal tracts supplemented with appropriate boundary conditions; a model for the glottal pulses generated by the vocal chords has to be input at the glottis area (Γ_G in Figure 4), and impedance and radiation conditions have to be respectively specified at the vocal tract walls Γ_W and mouth exit Γ_M , which are indicated later. Finally, collecting the time evolution of the acoustic pressure at a node close to the exit of the mouth and converting its values into an audio file, we will produce the sound of an /a/. It is to be noted that vowels are actually generated by acoustic resonance excitation of the air volume inside the vocal tract. The eigenfrequencies of the air resonating inside the vocal tract, which are known as formants in the voice and speech communities, will change from one vowel to another. Formants can be clearly appreciated as peaks in the so called vocal tract transfer functions (VTTF). These are computed from the quotient between the Fourier transform of the acoustic pressure at a node located close to the mouth exit, and the Fourier transform of the volume velocity corresponding to the glottal pulses being input at Γ_G . A VTTF for vowel /a/ is shown in the last row of the left column of Figure 4. The first two formants F1 and F2 have been highlighted as those are the ones which allow listening individuals to distinguish between vowel sounds (compare with the VTTF corresponding to an /i/ in the last row of the right column in Figure 4). Higher formants are responsible for other vowel sound finer features like timbre.

If we next focus on the generation of the diphthong /ai/, it is clear that we will have to transition from the situation in the left column in Figure 4 to the one in the right column. Some simplifications have been assumed. First, a linear interpolation from the geometry of the vocal tract of vowel /a/ to the vocal tract of vowel /i/ has been performed, though more accurate options exist, based on controlling the formant transition trajectories between vowels [44]. Second, we have simply imposed a zero pressure release condition $p = 0$ at the mouth exit Γ_M . This is a rough approximation to emulate radiation losses, which is known to shift the formants towards upper frequencies depend-

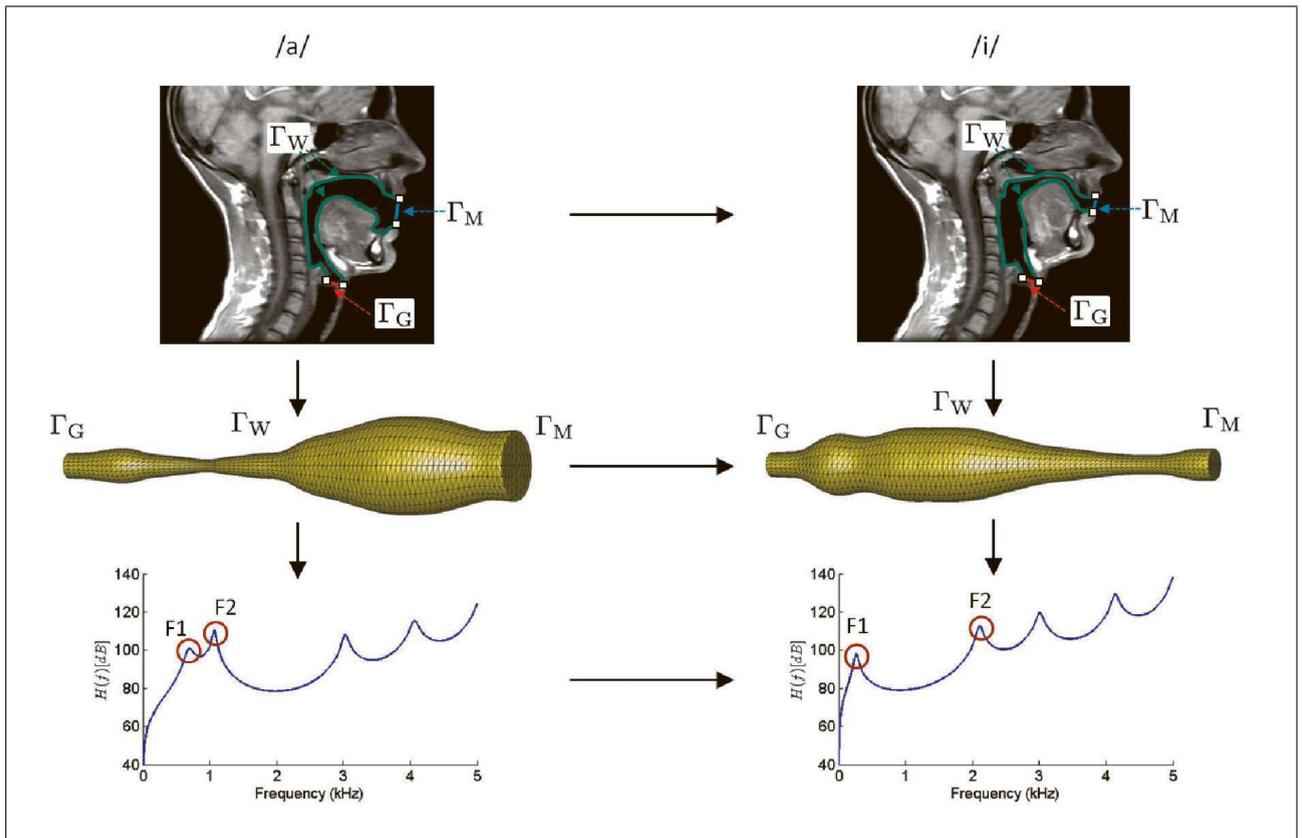


Figure 4. Sketch for the numerical production of vowels. From magnetic resonance imaging (MRI), data is obtained for the computation of vocal tract transfer functions (VTTF). The left column corresponds to vowel /a/ and the right column to vowel /i/. The diphthong /ai/ involves transiting from /a/ to /i/.

ing on the formant-cavity affiliation phenomenon [45, 46], as well as to decrease the formant bandwidths for mid-high frequencies (see e.g., [16] for a detailed discussion). A more precise option would consist in allowing acoustic waves to emanate from the mouth and radiate towards infinity. This would involve, for example, implementing a perfectly matched layer (PML) [47, 48, 49, 15] for the finite element approximation of the ALE wave equation presented in this work, or imposing a good enough performing absorbing boundary condition (see e.g., [50, 51]). This has been left for forthcoming developments.

To produce the diphthong /ai/ we have solved the ALE mixed wave equation (4) using the stabilized FEM approach described in the preceding Section 3, in the moving computational domain consisting of the transition from the vocal tract of /a/ in the left column of Figure 4, to the geometry of /i/ in the right column. The following conditions have been imposed on the vocal tract boundaries

$$\mathbf{u} \cdot \mathbf{n} = \mathbf{u}_g(t) \quad \text{on } \Gamma_G, \quad t > 0, \quad (35a)$$

$$\mathbf{u} \cdot \mathbf{n} = p/Z_w \quad \text{on } \Gamma_W, \quad t > 0, \quad (35b)$$

$$p = 0 \quad \text{on } \Gamma_M, \quad t > 0. \quad (35c)$$

The boundary velocity $\mathbf{u}_g(t)$ in (35a) stands for a train of glottal pulses generated using a model of the Rosenberg type [52]. In order to increase the naturalness of the produced diphthong these pulses have been enhanced introducing a pitch curve for the fundamental frequency, adding

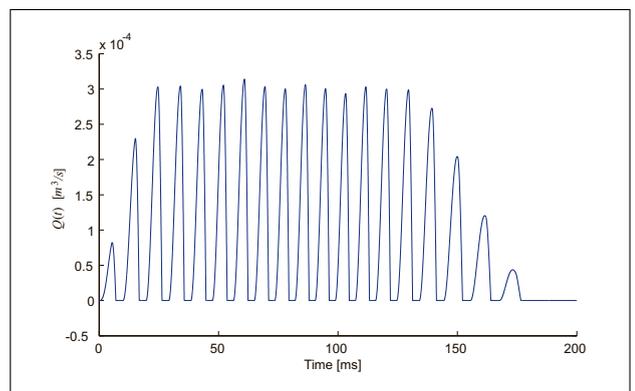


Figure 5. Train of glottal pulses introduced at the glottis to generate the diphthong /ai/.

some shimmer to the glottal pulse amplitudes and also applying a fade in/out to the global signal (see Figure 5). In (35b), Z_w stands for the wall impedance, which is related to the wall admittance coefficient, μ , by $\mu = Z_0/Z_w$, $Z_0 = \rho_0 c_0$ being the characteristic acoustic impedance. We have taken the standard value $\mu = 0.005$ which fairly approximates losses of vocal tract tissues [12]. Zero initial values have been taken for the acoustic pressure and acoustic particle velocity inside the vocal tract.

The moving vocal tract domain has been initially meshed using structured tetrahedral linear elements with

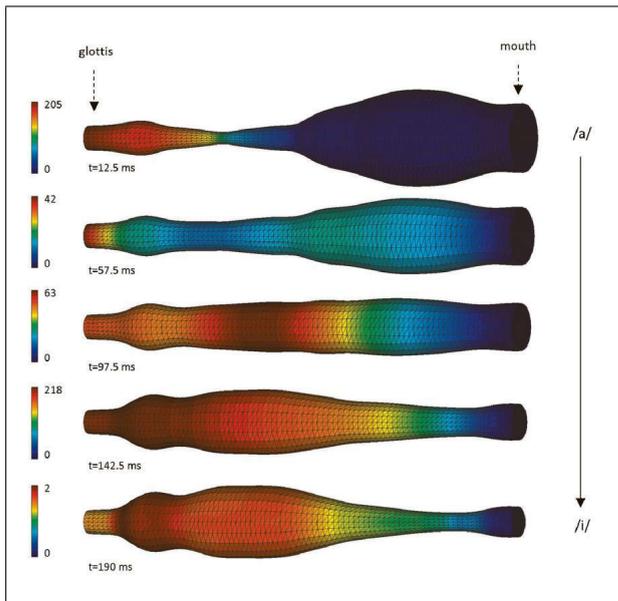


Figure 6. Snapshots at five different time instants showing the transition from vowel /a/ to /i/. The acoustic pressure in Pa at the boundaries of the vocal tract and its shape evolution can be appreciated.

initial size $h \simeq 0.002$ m and evolved according to the procedure described in Subsection 3.4. The simulation has lasted for 200 ms with a time step size of $\Delta t = 1.25 \times 10^{-5}$ s. As for the example in subsection 4.1, the latter is again much smaller than the numerical critical time step size. By the way, when choosing Δt one should also check that the time marching scheme does not introduce too much dissipation at the highest frequencies of the considered range ([0, 5] kHz in the present case). We have checked that variations in the VTTFs are less than 1% when taking a time step size of $\Delta t/2$. To which extent resorting to higher order numerical time schemes than BDF2 may allow using larger time steps to diminish the computational cost, or to keep the current cost but increasing the upper frequency range limit, will be worth exploring in the future. This shall be of special relevance for the more complex simulations mentioned above, that could account for wave propagation outwards the mouth. On the other hand, the standard value in vocal tract acoustics of $c_0 = 350$ m/s has been used for the sound speed whereas $\rho_0 = 1.21 \text{ kg/m}^3$ has been taken for the air density. Again, we have used the values $C_1 = C_2 = 0.01$ for the constants in the stabilization parameters (27). The diphthong simulation has lasted 8 hours in a serial computing system with processor Intel(R) Core(TM) i5 2.8 GHz.

In Figure 6, a sequence of five snapshots corresponding to the transition from vowel /a/ to /i/ is presented. The changes in vocal tract shape can be clearly appreciated. The first snapshot at $t = 12.5$ ms corresponds to the articulation of an /a/, whereas the last one at $t = 190$ ms corresponds to that of an /i/. The acoustic pressure values at the vocal tract boundaries for the selected time instants can be also appreciated. Different limiting values have been chosen for the color scales to better perceive the

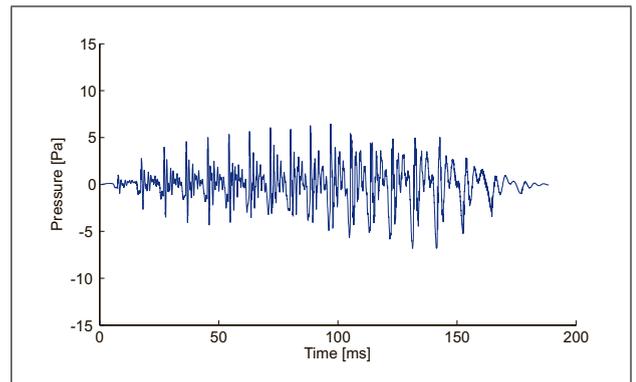


Figure 7. Time evolution of the acoustic pressure at a point close to the mouth exit. When converted to audio, it produces the sound /ai/.

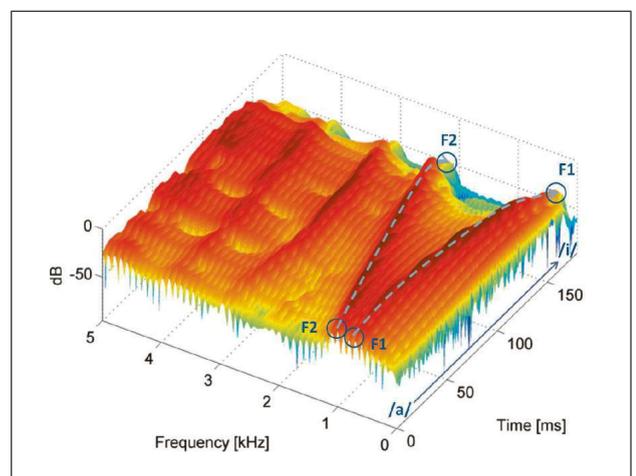


Figure 8. Spectrogram of the acoustic pressure at the node located close to mouth exit from Figure 7. Formant trajectories from /a/ to /i/ have been highlighted.

acoustic pressure distribution at the vocal tract for a particular time instant. In Figure 7 we show the time evolution for the acoustic pressure at a point close to the mouth exit. When transforming this plot into an audio file we get, as expected, the sound /ai/. Finally, in Figure 8 we show the spectrogram corresponding to the acoustic pressure in Figure 7. This has been computed using a hamming window with a frame width of 20 ms and an overlap of 1 ms. As usually done in speech processing, a pre-emphasis FIR filter with coefficients [1 -0.97] has been applied to enhance the visualization of the high frequency range. In the first curve of the spectrogram, corresponding to vowel /a/, we can clearly identify the two formants previously highlighted in the VTTF of /a/, in the last row of the left column of Figure 4. Similarly, the last curve of the spectrogram corresponds to vowel /i/, and the two formants identified in the VTTF of /i/ (last row of the right column of Figure 4) can also be clearly recognized. The trajectories of the two formants F1 and F2, which separate apart from /a/ to /i/, have been indicated in Figure 8 to better show their evolution.

Finally, it is worthwhile mentioning that the influence of the ALE terms in the mixed wave equation (4) is very small for the present diphthong example (probably irrelevant from a perceptual point of view because variations in the formant locations due to the ALE terms are less than 3%). This is so for the following two reasons: first the mesh velocity is small when compared to the sound speed and, second, the mesh distortion for the considered vocal tract geometries with radial symmetry mostly takes place in the direction perpendicular to the acoustic pressure gradient. These facts allow one to sample the mesh at a different rate than the acoustic waves to reduce the computational cost. Actually, in the present example the mesh is only updated every $10\Delta t$. It should be noted however, that the ALE terms may acquire more importance when generating diphthongs from more intricate MRI geometries and definitely, when producing more complex sounds like $/r/$. The proposed FEM formulation for the ALE mixed wave equation provides a proper mathematical framework to deal with the acoustics of varying vocal tract geometries.

5. Conclusions

A stabilized finite element method (FEM) has been proposed to solve the wave equation in mixed form in an ALE frame of reference. The method is based on the subgrid scale approach, and its good performance strongly relies on the proposed design for the matrix of stabilization parameters. The stabilized FEM allows us to use equal interpolations for the acoustic pressure and acoustic particle velocity, avoiding any locking effect. This has been tested by means of a two dimensional benchmark problem consisting of wave propagation inside a duct with a moving exit boundary.

On the other hand, the proposed approach has been applied to voice production, in particular to the generation of diphthongs. A general procedure has been presented for that purpose and diphthong $/ai/$ has been used as an example. Fairly good and promising results have been obtained. Future work will involve dealing with more realistic vocal tract geometries and permitting acoustic waves to emanate from the mouth and propagate outwards, to get more accurate solutions.

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